

Acoustic black hole project

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PROBLEM: OPTIMISE DAMPING IN STRUCTURAL PLATES

- Need for novel and efficient vibration damping
- Application for mass sensitive structural plates
- Study of convergence in classical bending plate theory (flexural waves)
- Damping of flexural waves by localised plate indentations causing zero reflection coefficient from free edges

PRACTICAL PROBLEMS

- The classical equations provide a physical insight into wave propagation
- A wave travelling into a quadratic power-law profile would increase in amplitude into non-linear regions, classical equations do not apply
- Any attempt to manufacture an ideal profile, leads to a tearing of the material, deviating from the profile and creating a free edge
- This free edge leads to reflection coefficients of 50-70 percent for flexural waves

PRACTICAL SOLUTION

- Krylov [2004, 2006]
- Application of viscoelastic damping to power-law profiles
- Elevated amplitudes and decreased wavenumber causes more efficient conversion of kinetic energy to heat in the damping layer
- As the plate thickness decreases, the composite damping (plate plus damping layer) rises significantly
- Numerical models produced of rectangular plates with damped quadratic profiles attached
- Experimental measurements of point mobility

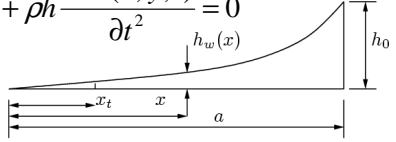
CONCLUSIONS

- Damped indentations show approximately double the vibration reduction compared to covering all plate with viscoelastic material
- Highly effective vibration damper
- Low reflection coefficient from free edges

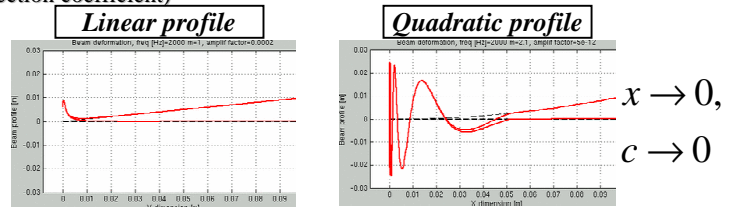
FLEXURAL WAVES IN IDEALLY TAPERED PROFILES

- A flexural wave travelling into a power-law profile (variable thickness) has a decreasing group speed and grows in amplitude
- Conservation of energy applies

$$\left(\nabla^2\right)\left(D\nabla^2\right)w(x, y, t) + \rho h \frac{\partial^2 w(x, y, t)}{\partial t^2} = 0$$

$$h(x) = \epsilon x^m$$


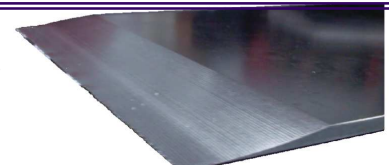
- Apply geometric acoustics approximation – Mironov [1988]
- Linear wedge (m=1), the wave reaches the end and reflects back (unity reflection coefficient)



- For the quadratic case, the equation of motion does not converge, the group speed asymptotically decreases to zero at the wedge end
- As it does not reach the end of the ideal profile, it cannot reflect and a zero reflection coefficient is achieved
- As the plate thins, the waves increase in amplitude, the group speed and the wavenumber decreases dramatically

EXPERIMENTAL AND NUMERICAL POINT MOBILITY

➤ O'Boy et al. *Journal of Sound and Vibration*, 329(11), 2010.



Plain plate	Plain plate with damping	Plain plate with damped wedge attachment
Frequency /kHz	Reduction in peak mobility amplitude /dB	Reduction in peak mobility amplitude /dB
2-4	-4.2	-8.5
4-6	-4.0	-11.0
6-8	-4.2	-6.0

